

# Non-isothermal cyclic fatigue in oscillating elasto-plastic structures with hysteresis

**Michela Eleuteri**

*Università degli Studi di Milano*

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Milano, June 5th, 2013

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- Theory of hysteresis operators powerful tool for solving mathematical problems in various applications (solid mechanics, material fatigue, ferromagnetism, phase transitions)

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- Influence of energy dissipation (due to material softening on the damage increase) taken into account - fatigue accumulation accelerated - a singularity (material failure) may develop in finite time
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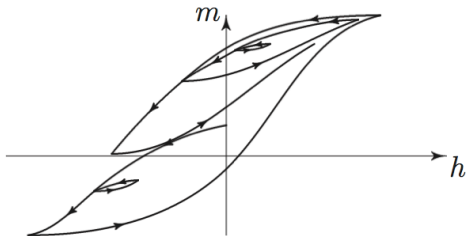
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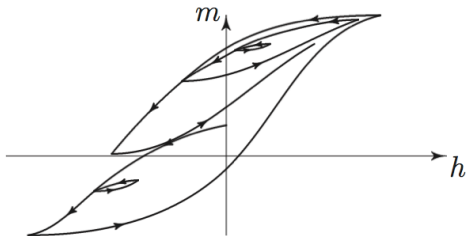


Typical hysteresis diagram in ferromagnetism ( $h$  magnetic field,  $m$  magnetization).

- Hysteresis present not only in ferromagnetism, but also in phase transitions, elastoplasticity, shape memory alloys, magnetostrictive and piezoelectric materials, economy, biology...

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# The stop model



Figure 1: The stop model.



# The stop model

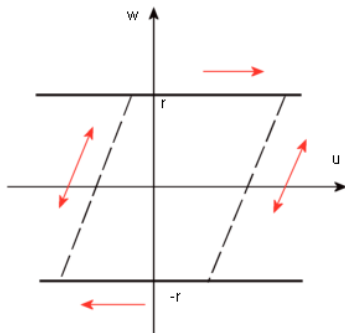


Figure 2: Hysteretic behaviour of the stop model.


# A classical hysteresis-type model for 1D elastoplasticity

- Given a parameter  $r > 0$ , a function  $\varepsilon : [0, T] \rightarrow \mathbb{R}$  and an initial condition  $\sigma^0 \in [-r, r]$
- We look for functions  $\sigma, \xi : [0, T] \rightarrow \mathbb{R}$  such that  $\sigma(0) = \sigma^0$  and

$$\sigma(t) + \xi(t) = \varepsilon(t)$$

$$|\sigma(t)| \leq r$$

$$\dot{\xi}(t)(\sigma(t) - \tilde{\sigma}) \geq 0 \quad \forall \tilde{\sigma} \in [-r, r].$$

- For all  $\varepsilon \in W^{1,1}(0, T)$  and  $\sigma^0 \in [-r, r]$ , the previous problem admits a unique solution  $\sigma \in W^{1,1}(0, T)$
- The map  $s_r : [-r, r] \times W^{1,1}(0, T) \rightarrow W^{1,1}(0, T)$ ,  $s_r[\sigma^0, \varepsilon] = \sigma$  is called stop or elasto-plastic element.  $s_r : [-r, r] \times C([0, T]) \rightarrow C([0, T])$
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# The multidimensional stop model

- Given a closed convex set  $Z$  including the origin in his interior  $0$  in a real separable Banach space  $X$ , a function  $\varepsilon : [0, T] \rightarrow X$  and an initial condition  $\sigma^0 \in Z$ , we look for functions  $\sigma, \xi : [0, T] \rightarrow X$  such that  $\sigma(0) = \sigma^0$  and

$$\sigma(x, 0) = Q_Z(\varepsilon(x, 0))$$

$$\sigma(t) \in Z$$

$$\left\langle \dot{\xi}(t), \sigma(t) - \tilde{\sigma} \right\rangle \geq 0 \quad \forall \tilde{\sigma} \in Z.$$

For all  $\varepsilon \in W^{1,1}(0, T; X)$  and  $\sigma^0 \in Z$ , the previous system admits a unique solution  $\mathfrak{s}_Z[\sigma^0, \varepsilon] = \sigma \in W^{1,1}(0, T; X)$ . The map  $\mathfrak{s}_Z : Z \times W^{1,1}(0, T; X) \rightarrow W^{1,1}(0, T; X)$  is continuous and admits a continuous extension  $\mathfrak{s}_Z : Z \times C([0, T]; X) \rightarrow C([0, T]; X)$

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- A classical hysteresis-type model for one-dimensional elastoplasticity was introduced by L. Prandtl and A. Yu. Ishlinskii
- In their model, the relation between (one-dimensional) strain  $\varepsilon$  and stress  $\sigma$  is given in the form of the so-called Prandtl-Ishlinskii operator

$$\sigma = \mathcal{P}[\varepsilon](t) = \int_0^\infty \mathfrak{s}_r[\varepsilon](t) \varphi(r) dr$$

for all  $\varepsilon \in W^{1,1}(0, T)$ . Here  $\varphi > 0$  is a nonnegative weight function not known a priori and  $\mathfrak{s}_r$  represents the **one-dimensional elastic-ideally plastic element or stop operator**, with the threshold  $r > 0$ .

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# The stop operators and their combinations

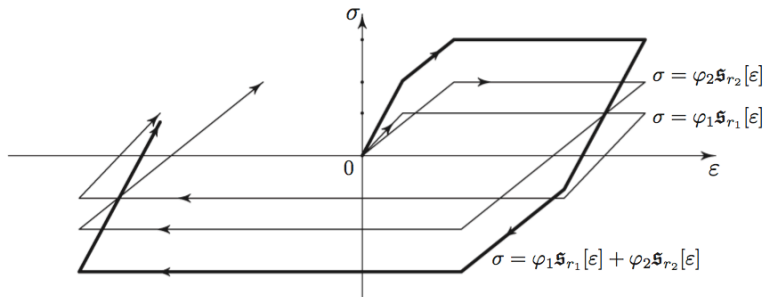


Figure 3: the stop operators and their combinations.

# Prandtl-Ishlinskii model v.s. Von Mises model

## The Prandtl-Ishlinskii model

- ☺ very imaginative and easily understood (superposition of many stops having different thresholds)
- ☺ **multi-yield**: describes gradual plasticization process
- ☺ the weight function  $\varphi$  is **not known a priori** and must be identified

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# A plate section with grey plasticized zone

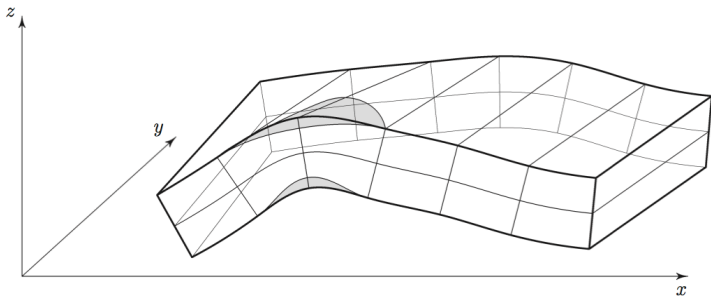


Figure 4: A plate section with grey plasticized zone.

# Motivation for the material fatigue

- ✎ It is well known that plastic deformations lead to energy dissipation and material fatigue
- ✎ Material fatigue is manifested by material softening, heat release, material failure in finite time
- ✎ Very important: take into account the effects of energy exchange between heat and mechanical energy, thermal stresses, and material fatigue
- ✎ In particular great importance for the applications: methods for estimating the lifetime of oscillating thermoelastoplastic structures under material fatigue
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# Literature concerning oscillating elastoplastic processes

- Classical approach

**G. Duvaut, J.L. Lions** (1972) - dynamical problem of plasticity of Prandtl-Reuss type is solved through the use of variational inequalities

- Quasistatic approach

**W. Han, B.D. Reddy** (1999); yield condition described by a sharp surface of plasticity - **M. Brokate, A.M. Khudnev** (2000); **M. Kuczma, P. Litewka, J. Rakowski, J.R. Whiteman** (2004); **O. Millet, A. Cimetiere, A. Hamdouni** (2003)

- $\Gamma$ -convergence approach

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# The system of PDEs

- The resulting system from the theory developed by Krejčí and coworkers (starting point of our new project) is the following:

$$\partial_{tt}w - \partial_{tt}\Delta w + \mathbf{D}_2^*\sigma = g,$$

$$\sigma = \mathbf{B}\varepsilon + \int_0^\infty \mathfrak{s}_{rZ}[\varepsilon](t) \varphi(r) dr$$

$$\varepsilon = \mathbf{D}_2 w$$

where  $\mathbf{D}_2$  is the second derivative operator  $(\partial_{xx}, \partial_{yy}, \partial_{xy})$  and  $\mathbf{D}_2^*$  is its adjoint

- We introduce a positive parameter  $\theta > 0$  indicating the absolute temperature and  $m(x, t) \geq 0$  a parameter which represents the material fatigue accumulated in the point  $x$  in the time interval  $[0, t]$
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# Coupling with the energy balance laws

- To the previous system we associate the **specific free energy**

$$\begin{aligned}\mathcal{F}[\theta, \varepsilon] &= c_V \theta (1 - \log(\theta/\theta_c)) + \frac{1}{2} \langle \mathbf{B}(m) \varepsilon, \varepsilon \rangle \\ &\quad + \frac{1}{2} \int_0^\infty \langle \mathfrak{s}_{rZ}[\varepsilon], \mathfrak{s}_{rZ}[\varepsilon] \rangle \varphi(\theta, r) dr - \beta(\theta - \theta_c) \langle \varepsilon, \mathbf{1} \rangle,\end{aligned}$$

with a constant specific heat  $c_V > 0$

- The **specific entropy** has the following form

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- from which, exploiting the well know relation  $\mathcal{F} = \mathcal{U} - \theta \mathcal{S}$  we get the following form of the **internal energy**

$$\begin{aligned}\mathcal{U}[\theta, \varepsilon] &= c_V \theta + \frac{1}{2} \langle \mathbf{B}(m) \varepsilon, \varepsilon \rangle \\ &\quad + \frac{1}{2} \int_0^\infty \langle \mathfrak{s}_{rZ}[\varepsilon], \mathfrak{s}_{rZ}[\varepsilon] \rangle (\varphi(\theta, r) - \theta \partial_\theta \varphi(\theta, r)) dr + \beta \theta_c \langle \varepsilon, \mathbf{1} \rangle.\end{aligned}$$

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- The **energy balance** can be written as

$$\partial_t \mathcal{U}[\theta, \varepsilon] + \operatorname{div} \mathbf{q} = \langle \sigma, \partial_t \varepsilon \rangle ,$$

where  $\mathbf{q}$  is the heat flux vector

- We derive the evolution law for the fatigue parameter which has to be compatible with the **Second Principle of Thermodynamics**, which we state in the form of the **Clausius-Duhem inequality**

$$\psi := \partial_t \mathcal{S}[\theta, \varepsilon] + \operatorname{div} \left( \frac{\mathbf{q}}{\theta} \right) \geq 0 ,$$

where  $\psi$  is the entropy production

- This implies that the **dissipation rate**

$$\begin{aligned} \mathcal{D} &= \langle \sigma, \partial_t \varepsilon \rangle - \partial_t \theta \mathcal{S}[\theta, \varepsilon] - \partial_t \mathcal{F}[\theta, \varepsilon] \\ &= -\frac{1}{2} \langle \mathbf{B}'(m) \varepsilon, \varepsilon \rangle \partial_t m + \int_0^\infty \langle \partial_t (\varepsilon - \mathfrak{s}_{rZ}[\varepsilon]), \mathfrak{s}_{rZ}[\varepsilon] \rangle \varphi(\theta, r) dr \end{aligned}$$

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# Evolution equation for the fatigue

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- The integral is non negative by virtue of the variational inequality which defines the stop operator [Details](#)
- The fatigue accumulation rate  $\partial_t m$  should be nonnegative. Hence, it suffices to assume that  $\mathbf{B}'(m)$  is a negative semidefinite matrix (softening!)
- The system will be complete by assuming the **linear Fourier law** between the heat flux and the temperature gradient

$$\mathbf{q} = -\kappa \nabla \theta,$$

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# The rainflow algorithm

- **Rainflow method for cyclic fatigue accumulation** - M. Endo (1968)
- The analysis of the rainflow algorithm in uniaxial processes has discovered a qualitative and quantitative relationship between accumulated fatigue and dissipated energy
- With an undamaged material, we associate fatigue value 0 , and 1 corresponds to total failure. Values between 0 and 1 quantify the degree of fatigue
- Experimental (decreasing!) curve  $n(b)$  (the so-called Wöhler line) determines how many closed cycles of amplitude  $b$  lead to total failure
- With each closed cycle of amplitude  $b$  we associate the contribution  $d(b) = \frac{1}{n(b)}$  of the individual cycle to total fatigue
- Therefore, the rainflow algorithm counts closed hysteresis loops in the loading history, and with each closed loop associates a number depending on its amplitude (the contribution of the loop to total damage) taken from the Wöhler line

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# The rainflow algorithm

- **Rainflow method for cyclic fatigue accumulation** - M. Endo (1968)
- The analysis of the rainflow algorithm in uniaxial processes has discovered a qualitative and quantitative relationship between accumulated fatigue and dissipated energy
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# The rainflow algorithm

- This corresponds to the mechanism of energy dissipation: the number associated with a closed loop is its area in this case
- Let a real loading process consist of a sequence of cycles with amplitudes  $b_j, j = 1, \dots, n$ . The Palmgren-Miner additivity rule states that the total fatigue can be computed as the sum of all individual contributions  $D_n = \sum_{j=1}^n d(b_j)$
- When the contribution of a closed cycle to total fatigue is evaluated, the cycle is removed from the history. At the end of the process, the remaining *residual* of the input signal contains no more closed cycles
- For a numerical treatment of large data sets it is important to assume that oscillations of very small amplitude cause no damage
- The rainflow method is then *stable with respect to small measurement errors independently of the number of cycles*
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# In multiassial processes?

- Drawback: the rainflow method is exclusively uniaxial - no closed cycles in the vector case
- In multiaxial loading processes, the concept of closed loop is meaningless, and no counterpart of the rainflow algorithm is known
- On the other hand, the notion of energy dissipation is independent of the experimental setting: experimental measurements at the point of material failure confirm strong temperature increase, which manifests an energy dissipation peak
- In fact, temperature tests are regularly used in engineering practice for damage analysis in high frequency regimes (e.g. in aircraft industry).

## Fundamental assumption

Assuming that the fatigue rate is proportional to the dissipation rate, and that the material parameters depend on the fatigue parameter

- Assumption realistic: plastic deformations are driven by moving dislocations and ruptures of interatomic connections, which at the same time dissipate energy, and reduce the cohesion of the solid

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# Evolution equation for the fatigue

$$\begin{aligned}\mathcal{D} &= \langle \boldsymbol{\sigma}, \partial_t \boldsymbol{\varepsilon} \rangle - \partial_t \theta \mathcal{S}[\boldsymbol{\theta}, \boldsymbol{\varepsilon}] - \partial_t \mathcal{F}[\boldsymbol{\theta}, \boldsymbol{\varepsilon}] \\ &= -\frac{1}{2} \langle \mathbf{B}'(m) \boldsymbol{\varepsilon}, \boldsymbol{\varepsilon} \rangle \partial_t m + \int_0^\infty \langle \partial_t (\boldsymbol{\varepsilon} - \mathfrak{s}_{rZ}[\boldsymbol{\varepsilon}]), \mathfrak{s}_{rZ}[\boldsymbol{\varepsilon}] \rangle \varphi(\boldsymbol{\theta}, r) \, dr\end{aligned}$$

- The integral is non negative by virtue of the variational inequality which defines the stop operator
- The fatigue accumulation rate  $\partial_t m$  should be nonnegative. Hence, it suffices to assume that  $\mathbf{B}'(m)$  is a negative semidefinite matrix (softening!)
- Fundamental assumption: proportionality between the rate of fatigue  $\partial_t m$  and the dissipation rate  $\mathcal{D}$

$$\left( \frac{1}{C(\boldsymbol{\theta})} + \frac{1}{2} \langle \mathbf{B}'(m) \boldsymbol{\varepsilon}, \boldsymbol{\varepsilon} \rangle \right) \partial_t m = \int_0^\infty \langle \partial_t (\boldsymbol{\varepsilon} - \mathfrak{s}_{rZ}[\boldsymbol{\varepsilon}]), \mathfrak{s}_{rZ}[\boldsymbol{\varepsilon}] \rangle \varphi(\boldsymbol{\theta}, r) \, dr$$

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# The model with phase transition

- **Motivation:**

- possibility to account also for decreasing fatigue rate (in view of engineering applications)
- the material can be partially repaired by local melting

- **How to achieve this goal:**

- Phase transition equation in the form of melting-solidification law

$$\alpha \chi_t \in -\partial_{\chi} \mathcal{F}[e, \theta, \chi] \quad \chi \in [0, 1]$$

$$\chi_0 \in [0, 1] \text{ some initial condition, } A(x, t) := \int_0^t \frac{1}{\alpha} \left( \frac{1}{\theta_c} (\theta - \theta_c) \right) (x, \tau) d\tau$$

$$(\chi_t - A_t)(z - \chi) \geq 0 \text{ for all } z \in [0, 1]$$

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- **How to achieve this goal:**

• extend the existing model by introducing a phase transition variable  $\chi$  (degree of melting) in the energy functional of the model (see [1, 2])

• possibly considering a sufficiently large time interval of observation (large enough to neglect a global solution of the constitutive PDEs system, but to be able

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- **How to achieve this goal:**

- account for phase transition in the model
- as material fatigue and  $\chi$  degree of melting
- the evolution of  $\chi$  in the model can be obtained by properly considering as a boundary value problem of parabolic type the evolution of the degree of melting of the component. The evolution of  $\chi$  is then given by

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- account for phase transition in the model
- $m$  material fatigue and  $\chi$  degree of melting
- the time of failure of the material can be shifted
- possibly considering a sufficiently large time interval of observation (usual engineering viewpoint) a global solution of the corresponding PDEs system can be found

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$$\chi_0 \in [0, 1] \text{ some initial condition, } A(x, t) := \int_0^t \frac{1}{\alpha} \left( \frac{L}{\theta_c} (\theta - \theta_c) \right) (x, \tau) d\tau$$

$$(\chi_t - A_t)(z - \chi) \geq 0 \text{ for all } z \in [0, 1]$$

# The model with phase transition

## ● Motivation:

- possibility to account also for decreasing fatigue rate (in view of engineering applications)
- the material can be partially repaired by local melting

## ● How to achieve this goal:

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- $m$  material fatigue and  $\chi$  degree of melting
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$$\chi \in \mathfrak{s}_{[0,1]}[\chi_0, A]$$

$\mathfrak{s}_{[0,1]}$  is a shifted stop

# Thermodynamical consistency

- If we introduce  $\mathcal{F}[\varepsilon, \theta, \chi]$  **specific free energy**,  $\mathcal{S}[\varepsilon, \theta, \chi]$  **specific entropy** and  $\mathcal{U}[\varepsilon, \theta, \chi]$  **internal energy** we are able to show that the first and second principles of thermodynamics are satisfied

$$\frac{\partial}{\partial t} \mathcal{U}[\varepsilon, \theta, \chi] + \operatorname{div} \mathbf{q} = \langle \sigma, \varepsilon_t \rangle \quad (\text{energy conservation})$$

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- **Evolution equation for  $m$ :**

$$(C + \mathcal{K}[m, \theta, \varepsilon]) m_t = -h(\chi_t) + \mathcal{D}[m, \theta, \varepsilon]$$

- where

$$\mathcal{D}[m, \theta, \varepsilon] := \int_0^\infty \varphi(m, \theta, r) \langle \mathbf{K} s_{rZ}[\varepsilon], (\varepsilon - s_{rZ}[\varepsilon])_t \rangle dr$$

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